



## Structural Static Analysis of Connecting Rod

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### **Abstract-**

*The principal of connecting rod is to convert linear motion of piston to reciprocating motion of crankshaft. It is the main component of internal combustion (IC) engine. The connecting rod takes more maximum stress compare to other components. so stress analysis in Connecting rod is Very important. In this study, detailed load analysis was performed on connecting rod, followed by finite element method in hypermesh and abaqus medium. The maximum stresses in different parts of connecting rod were determined by abaqus.in this work we are investigated the stress, strain deformation.*

### **I.INTRODUCTION**

The main purpose of connecting rod is to convert linear motion of piston to reciprocating motion of crankshaft. It is the main component of internal combustion (IC) engine. During its operation different stresses are acting on connecting rod. Basically the Connecting rod is the link between the piston and the crank. And is responsible to transmit the tension and compression from the piston pin to crank pin, thus converting the reciprocating motion of the piston to rotary motion of the crank. This tendency in vehicle construction led the invention and implementation of quite new materials which are light and meet design requirements.

There are various types of materials and production methods used in the creation of connecting rods. The most common types of

connecting rods are aluminum and steel. The most common type of manufacturing processes is powder metallurgy, casting, forging. Connecting rod is among large volume production component in the I.C engine. They are different types of materials and production methods used in the creation of connecting rods. The major stresses induced in the connecting rod are of axial and bending stresses in operation. The axial stresses are produced due to pressure (compressive only) and the inertia force arising in account of reciprocating (both tensile as well as compressive), whereas bending stresses are produced due to the centrifugal effects. It consists of a long shank, a small end and a big end. Connecting rod is subjected to a complex state of loading. The influence of compressive stress is more in connecting rod due to gas pressure and whipping stress. As improving a method for providing high productivity with a lower production cost, generally as the forces generated on the connected rod are by weight and combustion of fuel inside cylinder acts upon piston and then on the connecting rod, which results in both the bending and axial stresses.

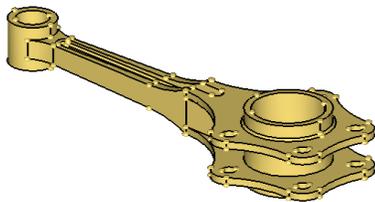


**Fig 1.** Schematic model of master connecting rod.

## II. METHODOLOGY

Connecting rods are used in numerous situations, most commonly in the engines of automobiles. The purpose of a connection rod is to provide fluid movement between pistons and a crankshaft. Connection rods are widely used in vehicles that are powered by internal combustion engines. All automobiles that use this type of engine employ the use of connecting rods. Even construction equipment like bulldozers use internal combustion engines, thus requiring connecting rods.

Master connecting rod carries one or more ring pins to which are bolted and much smaller big end of slave rod on other cylinder. Advantage of this is, it is having strokes larger than the normal connecting rod.



**Fig 2.3D** model of master connecting rod

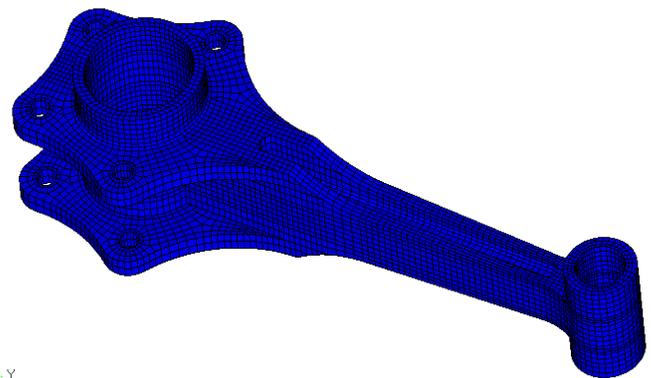
Meshing of MCR is carried out in HYPERMESH. Here we are going to make solid mesh with the element size 3mm.

### A. Material Properties for Master Connecting Rod

**Table 1.** Properties of materials.

Sl.no	Material property	values
1	Density	$7.89e-9$ N/mm <sup>2</sup>
2	Young's modulus	$2.1e5$ N/mm <sup>2</sup>
3	Poisson's ratio	0.3

### B. Boundary Conditions



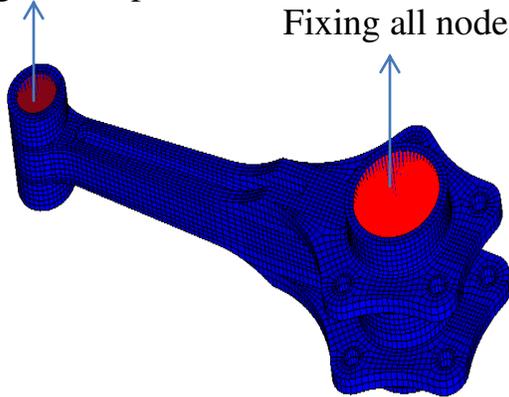
**Fig 3.** Meshed model of MCR

- Quality checks
  - ✓ Warpage=30
  - quad faces
  - ✓ Min angle=45
  - ✓ Max angle=155

- ✓ Jacobian=0.5
- Tria faces
- ✓ Min angle=11
- ✓ Max angle=125

Applying load at point

Fixing all nodes

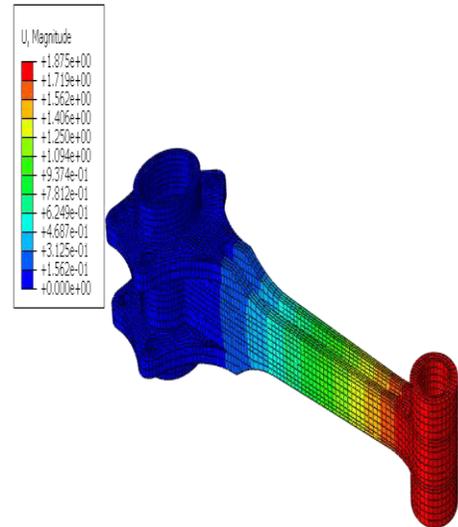


x

**Fig 4. Applied boundary condition**

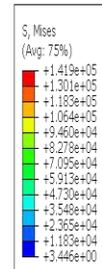
### III. RESULTS AND DISCUSSIONS

#### 1. Deformation Result



008: master.odb Abaqus/Standard 6.12-1 Fri Dec 20 00:04:05 India Standard Time 2013

Step: Step-1  
Increment: 6; Step Time = 1.000  
Primary Var: U, Magnitude  
Deformed Var: U, Deformation Scale Factor: +4.273e-01



008: master.odb Abaqus/Standard 6.12-1 Fri Dec 20 01:06:10 India Standard Time 2013

Step: Step-1  
Increment: 6; Step Time = 1.000  
Primary Var: S, Mises  
Deformed Var: U, Deformation Scale Factor: +4.273e-01

Fig 5. Deformation result

2. Stress Results

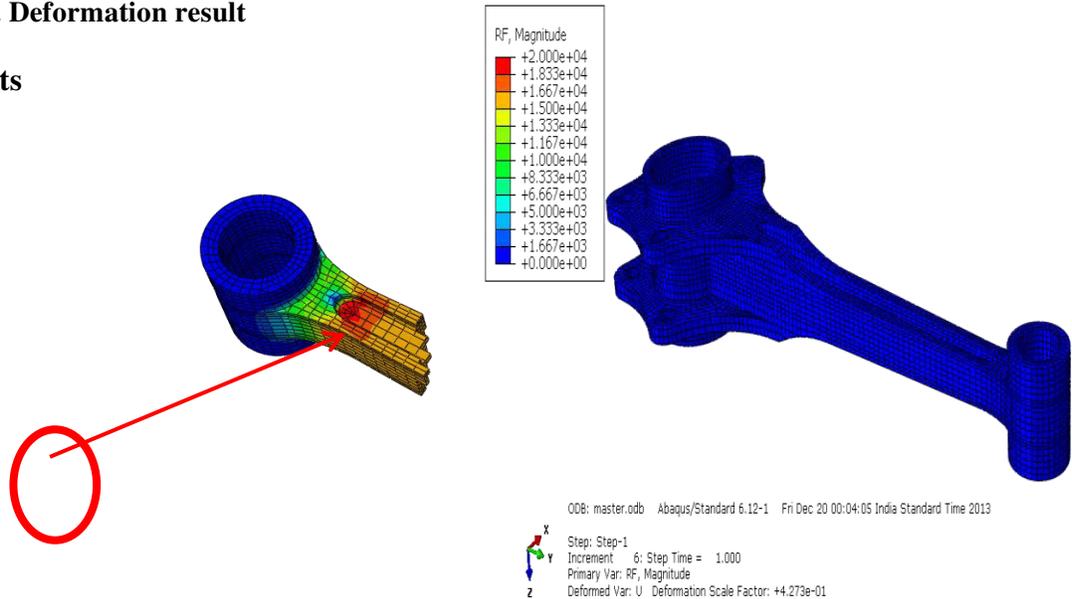
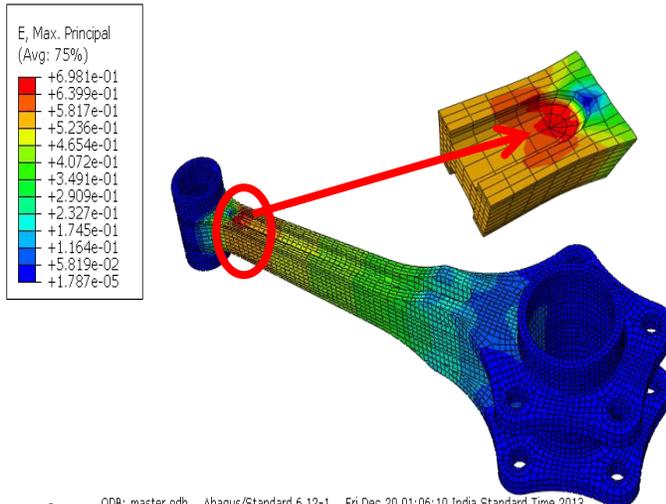
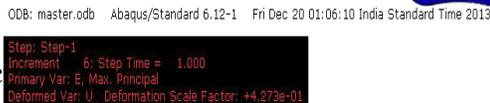


Fig 6. Stress Results

3. Strain Results



4. Reaction Results



Probe Values

Field Output... Step: 1, Step-1 Frame: 6

Field output variable for Probe: RF, All direct

Probe Values: Select from viewport (selected) Key-in label Select a display group Enter labels to store values in the table below.

Probe: Nodes Components: All Part instance: PART-1-1 Node labels: 24484

Part Instance	Node ID	Orig. Coords	Def. Coords	Attached elements	RF, RF1	RF, RF2	RF, RF3
PART-1-1	24484	5.61556e-08, -2	5.61556e-08, -2		9.11216e-1	-20000	0.000318941

Note: Click on respective check button to annotate values in viewer

Write to File... Cancel

The above picture shows that reaction force is equal to applied force but in the opposite sign, i.e Newton's third law.

Fig 6. Reaction Results.



#### IV. Conclusions

From this analysis of connecting rod we come to know the conclusion that,

1. Connecting rod is subjected to tensile load.
2. Stress and strain are more at small end of connecting rod.
3. Stress value crossing tensile strength so that design is not safe, we have to increase the dimension of connecting rod.

#### Reference

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